

IN THE CLAIMS

Please amend the claims as follows:

1. (Cancelled)
2. (Previously Presented) Sealing arrangement according to claim 10 wherein in an unpressurized state, an inner radius of the U-cup seal decreases continuously from the low-pressure side towards the inner sealing lip.
3. (Previously Presented) Sealing arrangement according to claim 2, wherein in the unpressurized state, the inner radius of the U-cup seal conically decreases from the low-pressure side towards the inner sealing.
4. (Previously Presented) Sealing arrangement according to claim 10 wherein in an unpressurized state, the abutment surface of the U-cup seal is, in part, convex.
5. (Previously Presented) Sealing arrangement according to claim 10 wherein the U-cup has an outer concave surface adjacent a groove bottom in the unpressurized state.
6. (Previously Presented) Sealing arrangement according to claim 10 wherein in an unpressurized state, an outer radius of the U-cup seal increases continuously from the low-pressure side towards the outer sealing lip.
7. (Previously Presented) Sealing arrangement according to claim 10 wherein the U-cup seal has an outer

surface facing away from the movable piston and that in an unpressurized state, an outer edge of the U-cup seal is formed convex in a transition region between abutment surface and outer surface.

8. (Previously Presented) Sealing arrangement according to claim 10 wherein the U-cup seal has an inner surface facing the movable machine part (1), with calotte shells as a microstructure.

9. (Cancelled)

10. (Currently Amended) Sealing arrangement comprising a U-cup of a viscoplastic synthetic material, a stationary machine part, and a translatory movable machine part with an outer radius R , wherein the U-cup is disposed as a contacting joint under radial prestress between the stationary machine part and the movable machine part in a profiled section of the stationary machine part, wherein the U-cup has a radially outer and a radially inner sealing lip on ~~the~~ a high-pressure side, wherein the stationary and the movable machine parts are separated on ~~the~~ a low-pressure side by ~~a sealing gap of~~ a sealing gap width B , wherein an abutment surface of the U-cup abuts a radially oriented region of the profiled section on the low-pressure side, wherein the U-cup has an inner radius and an outer radius, wherein both in the unpressurized state and in the pressurized state, the inner radius of the U-cup in the region of the abutment surface is larger than the sum of R and B and wherein the U-cup comprises an inner surface facing the movable machine part, wherein the inner surface comprises several lubrication bore reliefs formed as

recesses in the inner surface of the U-cup causing hydraulic liquid disposed on a surface of the translatable movable machine part to be dragged from a low pressure region to a high pressure region upon translatable movement of the movable machine part, wherein the recesses each extend in an axial direction from the low pressure side N of the U-cup towards the inner sealing lip, and the radial depth of the individual recesses decreases from the low-pressure side N of the U-cup towards the inner sealing lip, the recesses being spaced apart from the moveable machine part in an unpressurized state.